

# **The NET Power Cycle and the Combustor and Turbine Development**

**March 2013**

**Toshiba Corporation  
Power Systems Company**

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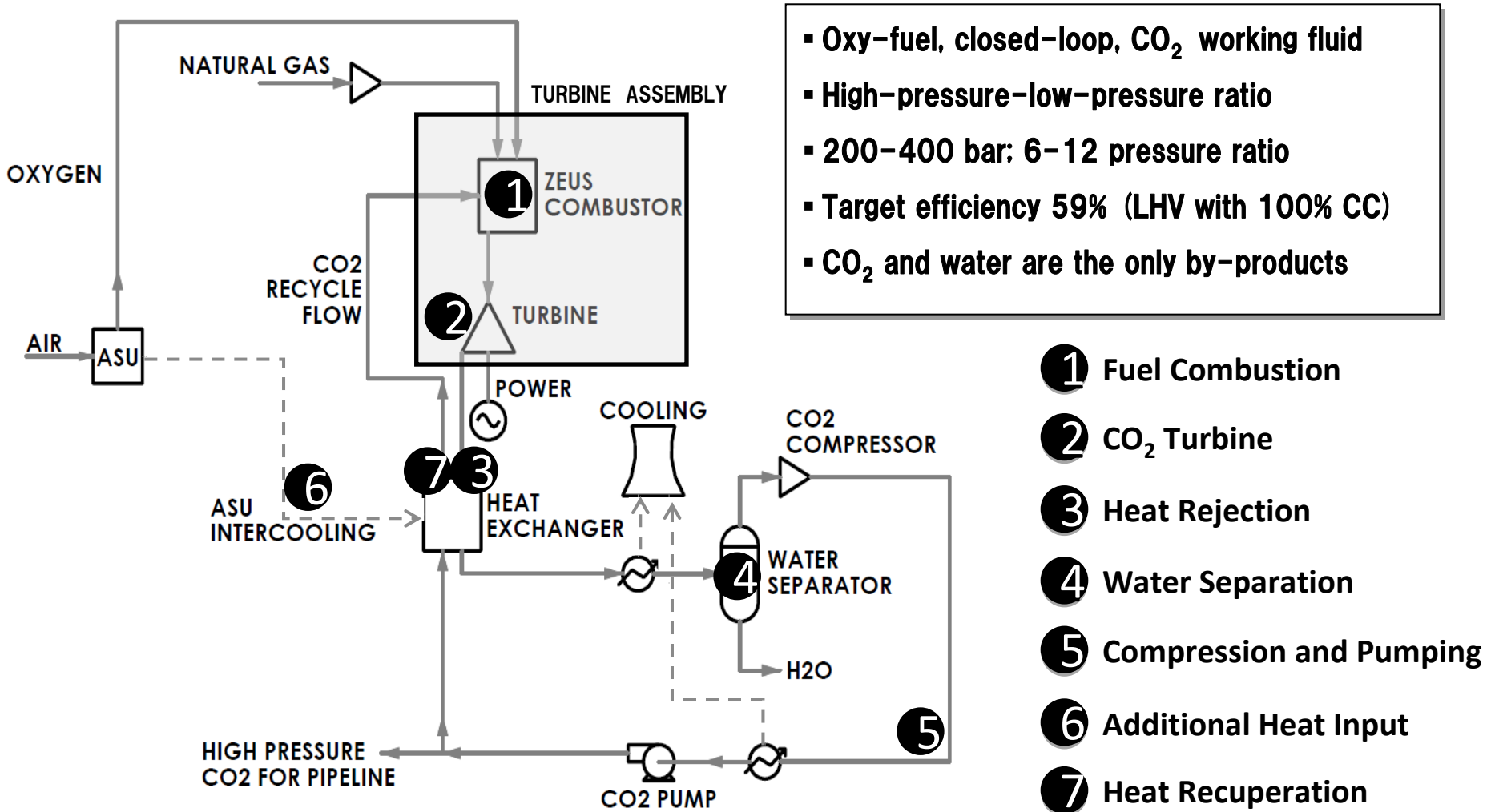
**1. Platform Cycle & Other Applications**

**2. Schedule**

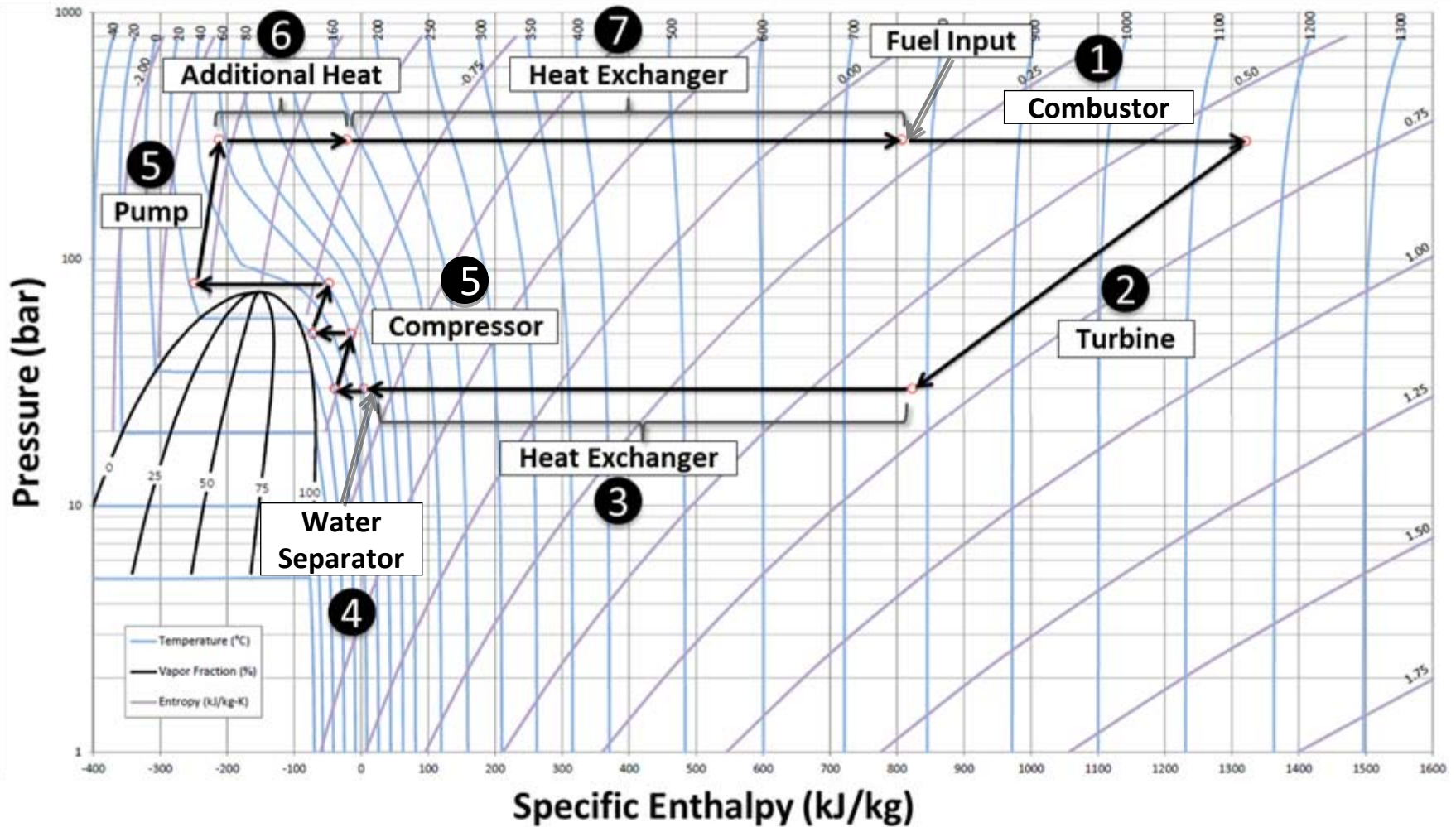
**3. Concept of Turbine Design and Present Status**

**4. Combustor Design and Rig test**

# Natural Gas Cycle: The Platform



# Pressure and Enthalpy Diagram



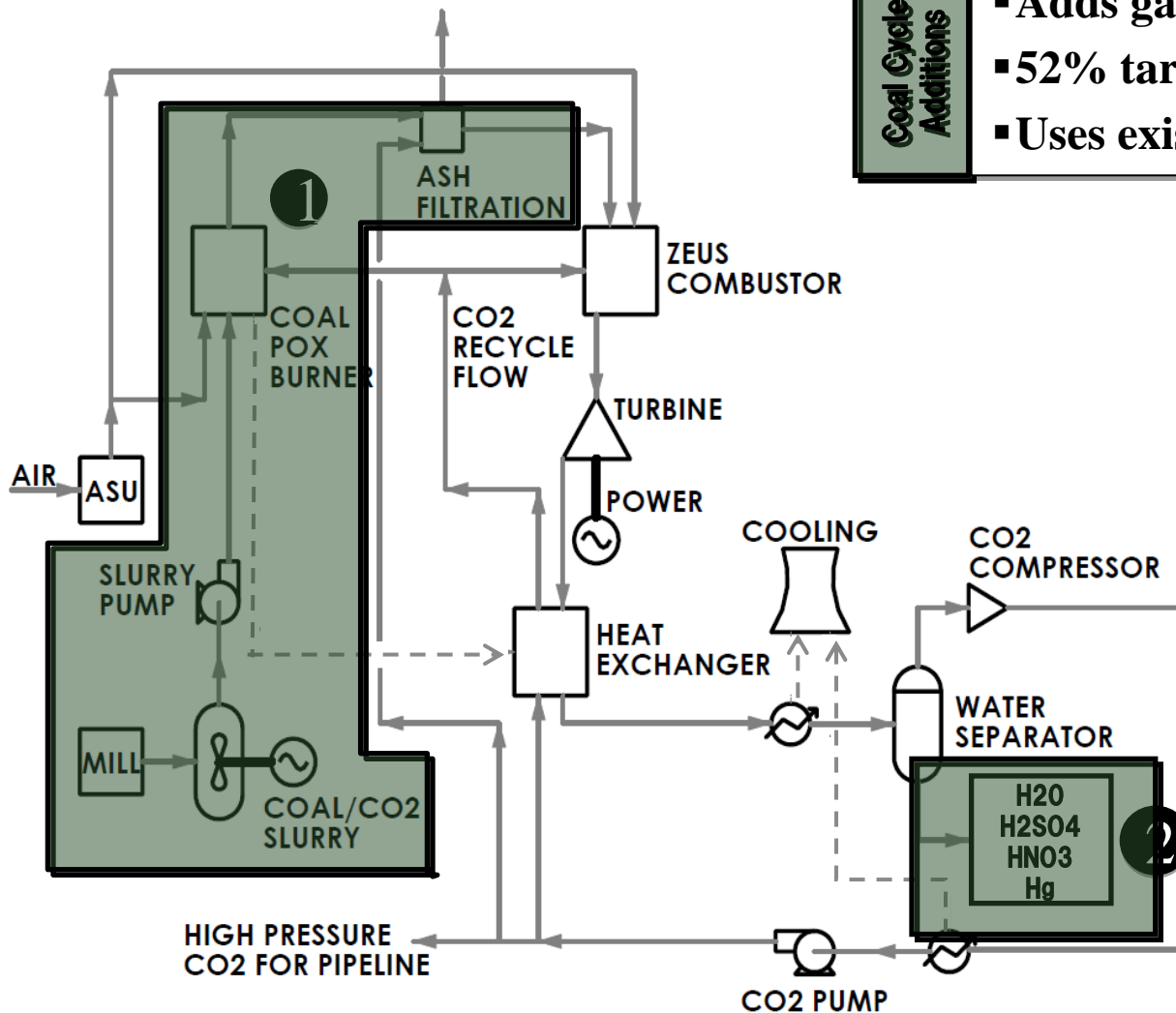
# NET Power Platform Target Efficiency

Natural Gas Platform Target Efficiencies (100% CO <sub>2</sub> Capture at 300 bar)		
Energy Components	HHV	LHV
Gross Turbine Output	75%	83%
CO <sub>2</sub> Compressor Power	-11%	-12%
Plant Parasitic Power (primarily ASU)	-11%	-12%
Net Efficiency	53%	59%

# Coal Application

## Coal Cycle Additions

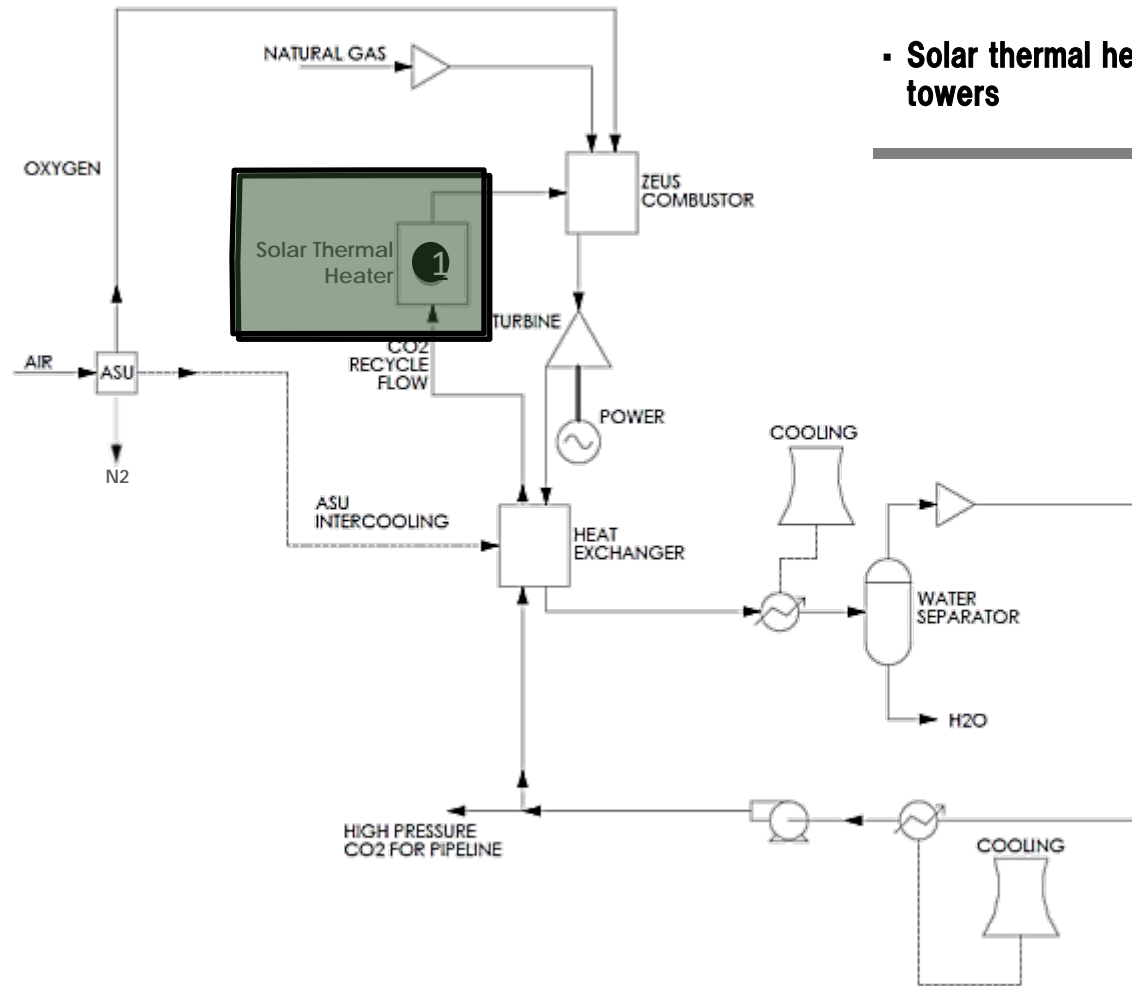
- Adds gasification & cleanup
- 52% target eff. (LHV with 100% CC)
- Uses existing gasification technology



- 1 Coal Gasification
- 2 Impurity Cleanup

# Direct Solar Integration

- Year-long efficiency at 74% in US Southwest
- Solar thermal heat most likely in the form of solar power towers

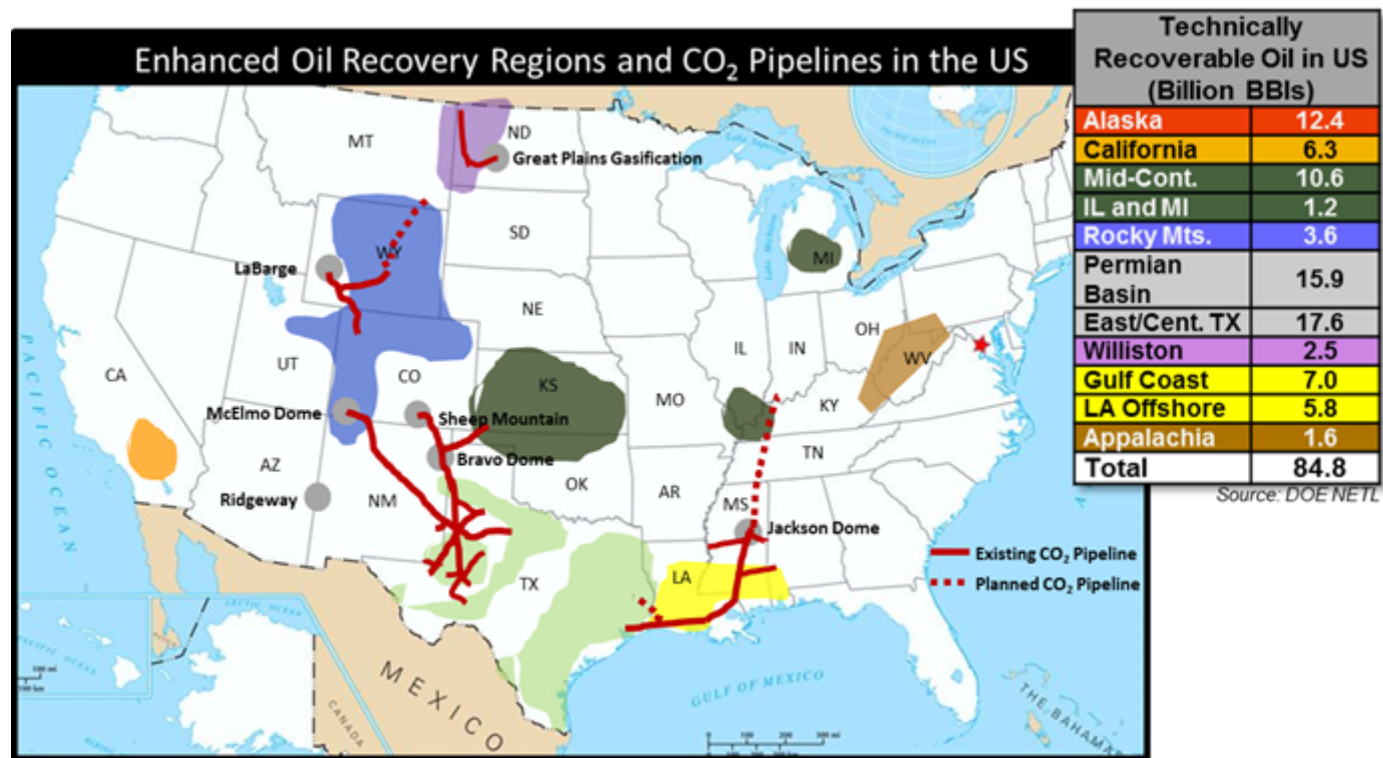


① Solar-thermal heat source



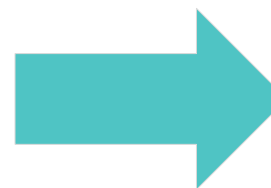
# EOR Market: A Large Financial and CO<sub>2</sub> Storage Opportunity

85 billion barrels technically recoverable in the US; industry is tethered to current pipeline and geologic CO<sub>2</sub> infrastructure



470-1,000 billion barrels of oil technically recoverable globally

GLOBAL CO <sub>2</sub> EOR POTENTIAL	
Region	Billion Bbls
Middle East	230
Russia	78
United States	85
S. America	32
Asia Pacific	18
Europe	16
Africa	15



Assuming a plant size of 550 MW, this need would support the CO<sub>2</sub> production from 1382 NET Power gas plants (691 coal).



# Four Way Agreement and Commercial Relationships

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**NET Power\***

Inventor and developer of the technology.  
Responsible for overall project development,  
systems engineering and commercial  
development

**Toshiba\***

Developing the turbine and combustor

**Goodwin Steel\***

**CB&I (Shaw\*)**

Provided substantial investment in this  
project and performing EPC services.

**Exelon**

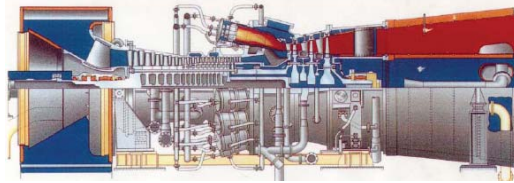
Assisting with the siting, permitting, and  
commissioning of the natural gas demo facility;  
providing operations and maintenance support.

\*Note; UK grant was awarded for the development.

# Necessary Technology for NET-Power Turbine

## Gas Turbine Technology

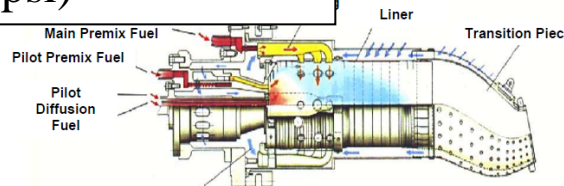
1100-1500C (2012-2730F)



Working fluid; CO<sub>2</sub>  
Pressure; 2MPa  $\Rightarrow$  30MPa  
(290psi  $\rightarrow$  4350psi)

## Combustor Technology

1100-1500C



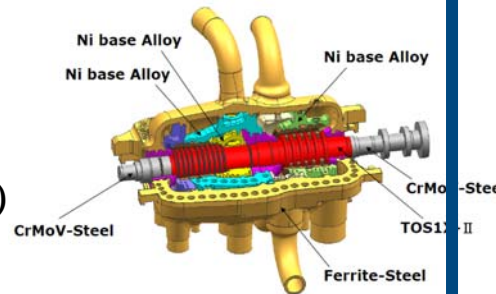
Working fluid; CO<sub>2</sub>  
Pressure; 2MPa  $\Rightarrow$  30MPa

## Steam Turbine Technology

USC & A-USC

Pressure; 24-31MPa (3500-4500psi)

Temperature; 600-750C (1112-1382F)



Temperature  $\Rightarrow$  1150C (2110F)

***Temperature; E-Class  
Pressure; USC & A-USC***

**Turbine & Combustor for  
Net Power**

Temp. 1150C (2110F)  
Press. 30MPa (4350psi)

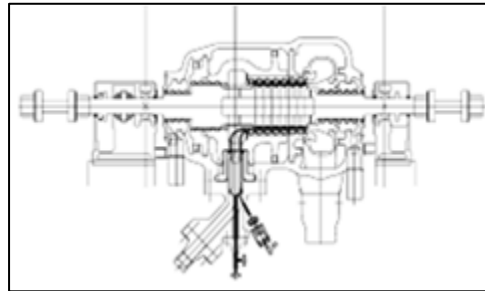
***Toshiba is the only  
company that  
manufactured commercial  
turbine of 31Mpa main  
steam pressure.***

***Toshiba has been keen on  
A-USC development.***

# Turbine for 25MWe Demo Plant

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- A) Intended to be a scale model of commercial turbine (250MWe) as much as possible
- B) Rotational speed is 6000rpm and connected to compressor and reduction gear
- C) Double shell configuration
- D) Rotors are welded together
- E) Single can type combustor for 25MWe turbine



## Materials

Rotor; Ni base forging and CrMoV forging are welded together

Casings; Ni base casting for high temperature part

CrMoV casting for lower temperature part

Blades; Ni base casting

All the necessary materials have been already developed.

Purchase orders for long-lead materials will be placed soon.

# R&D results of Ni base Forging and Ni base Casting

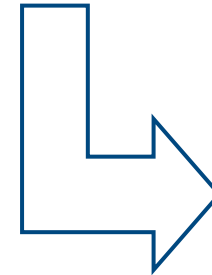
## - Make the best use of R&D results for A-USC -



First trial forging for TOS1X was completed (above photograph)  
Second trial forging for rotational test will be manufactured soon.



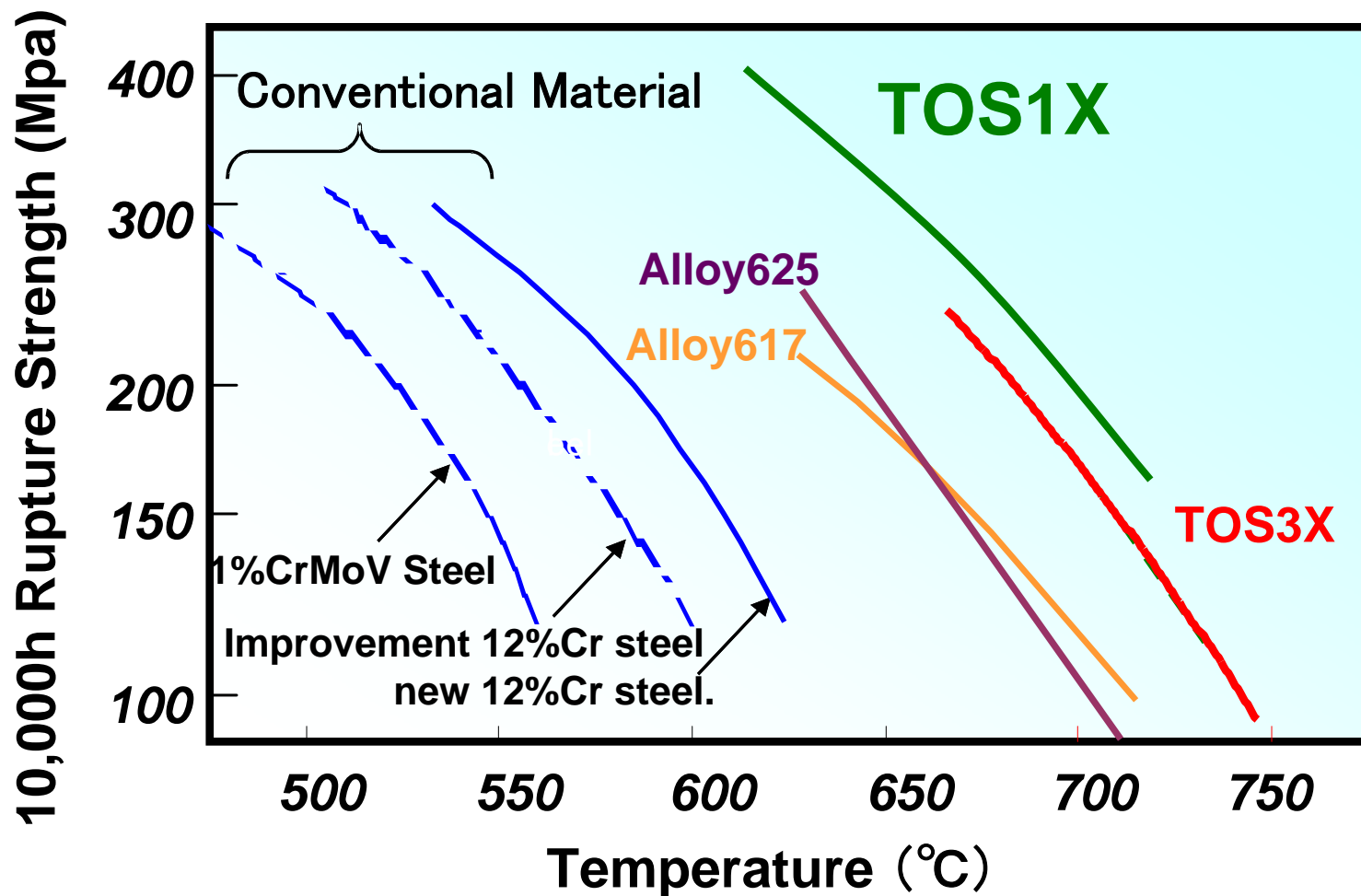
Alloy625  
Trial Inner casing for A-USC



TOS3X Test Piece

Two candidates, Alloy 625 and TOS3X, are available for S-CO<sub>2</sub> Turbine

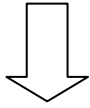
# Comparison of Creep Rupture Strength



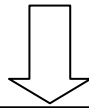
Necessary materials are already applicable

# Cooling Design

High Temperature

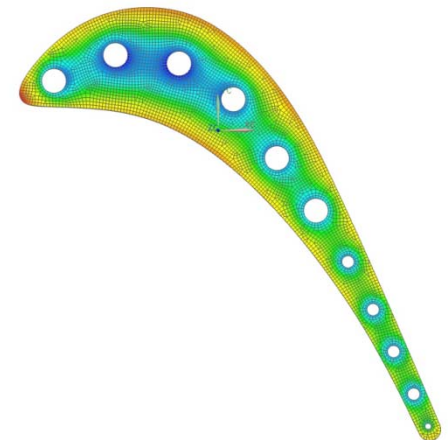


- Needs cooling both for nozzles and moving blades
- However, very complicated cooling technology is not necessary because the temperature is not extremely high compared with cutting-edge gas turbines.



Both mean temperature and local temperature satisfy design criteria thanks to two contributor

- Convection cooling by cold CO<sub>2</sub>
- Thermal barrier coating

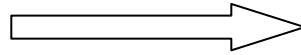


Temp. Contour



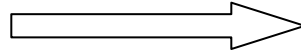
# Concept of Combustor Design

No NO<sub>x</sub> Emission



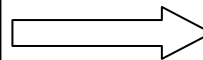
No need of complex pre-mix technology  
Simple Diffusion Flame can be used

High Pressure

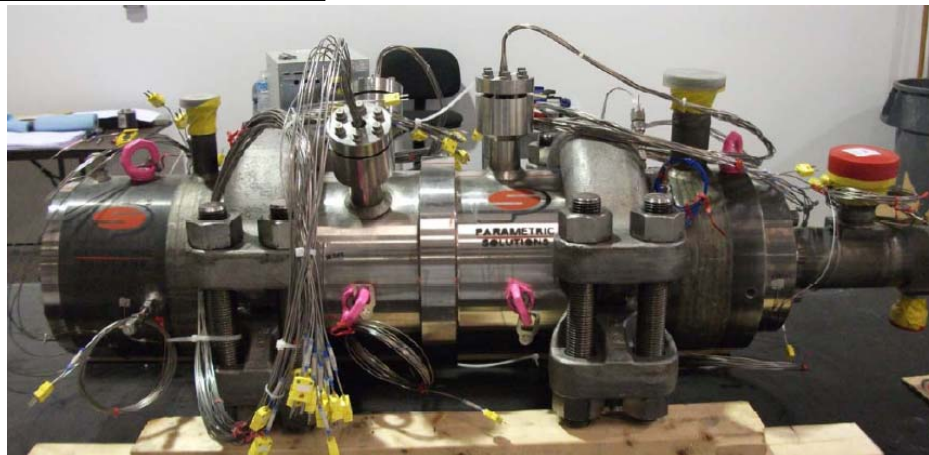


Thick Wall Casing against high Pressure

Rather moderate temperature compared with gas turbines



Experienced Cooling Scheme

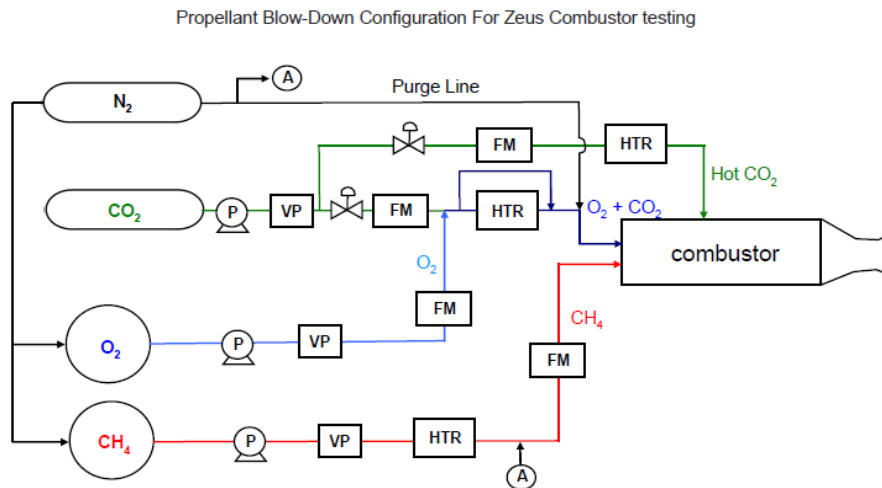


5MWt Rig Test Combustor

Rather simple combustion and cooling are expected , yet verification under high pressure using a rig test combustor is necessary.

# Present Status of Combustor Development

- ✓ First Ignition was successfully done at the middle of January using test facility in U.S.A.
- ✓ Phase 1 Test up to 5Mpa has been completed.
- ✓ All the test data was carefully checked and evaluated.
- ✓ Stable flame was confirmed enabling us to proceed to higher pressure test (Phase 2).
- ✓ Modification of facility is being done for Phase 2 test.
- ✓ Design of the combustor for 25MW Demo plant will be synchronized with Turbine Development.



Rig Test Combustor at the Test Stand



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# END